

Slide 1

LECTURE NOTES  
ON  
**REFRIGERATION & A/C**  
CE C471 / ME C461  
**AIR [GAS]CYCLE REFRIGERATION**  
**PART-2**  
**BY**  
**DR.A.M.SURENDRA KUMAR**

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Slide 2

**INTRODUCTION**

- Air is used as working fluid.
- No change of **phase** through out.
- Heat carrying capacity/kg of air is very **small** compared with other refrigerant systems.
- **High** pressure air readily available in the Aircraft .
- **Low** equipment weight.

• **Basic elements:**  
1. **Compressor** 2. **Heat exchanger** 3. **Expander** 4. **Refrigerator**

**Open system** : The air used in the refrigerator is thrown into the atmosphere.

**Closed system**: Air used is recirculated

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Slide 5

- To increase C.O.P.,  $T_2$  should kept low.
- But cannot be reduced below  $25^\circ\text{C}$  –Atmospheric Temp.
- $T_1$  should be kept high.
- But cannot be increased above  $0^\circ\text{C}$ .
- It is the required temperature.

If say  $T_1 = 0^\circ\text{C}$  which is the required temperature for the production of ice then

$$T_1 \text{ in winter} < T_2 \text{ in summer}$$

so that for the same required temperature, Carnot refrigerator works more efficiently in winter than in summer because

C.O.P. in winter > C.O.P. in summer.

Assuming  $40^\circ\text{C}$  and  $20^\circ\text{C}$  are the average temperatures in summer and winter, then

$$\frac{0 + 273}{20 - 0} > \frac{0 + 273}{40 - 0}$$

(C.O.P. in winter)  $\frac{273}{20}$  > (C.O.P. in summer)  $\frac{273}{40}$

This explains the effect of atmospheric temperature on C.O.P. of Carnot refrigerator used for particular purpose.

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Slide 6

**Example:**

If Carnot Refrigerator is used for making ice, (Required temp is 0°C) and for A/C plant (required temp. to be maintained is 20°C) in summer when the atmospheric temp. is 40°C, then the Carnot refrigerator used for A/c plant works more efficiently than the Carnot refrigerator used for making ice because C.O.P of Carnot refrigerator used for making ice

$$\frac{20 + 273}{40 - 20} > \frac{0 + 273}{40 - 0}$$

$$\frac{293}{20} > \frac{273}{40} \text{ as } \frac{586}{40} > \frac{273}{40}$$

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Slide 7

**ADVANTAGES OF AIR –REFRIGERATION SYSTEMS**

1. As the air is easily available compared with the other refrigerant, it is **cheap**.
2. The air used is **non-flammable**, so there is no danger of fire as in NH<sub>3</sub> machine.
3. The weight of the air refrigeration system / T.R is quite **low** compared with the other refrigeration systems which is one of the major causes selecting this system in air craft.

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Slide 8

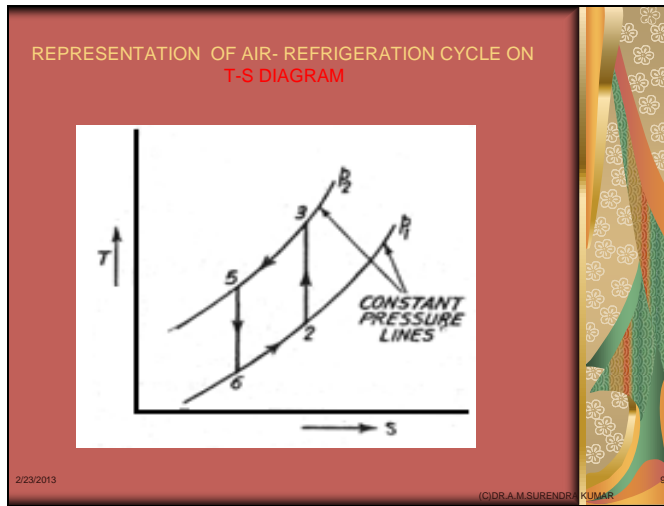
**BELL-COLEMAN REFRIGERATOR**

Fig. Closed cycle air-refrigerator working on Bell-Coleman cycle.

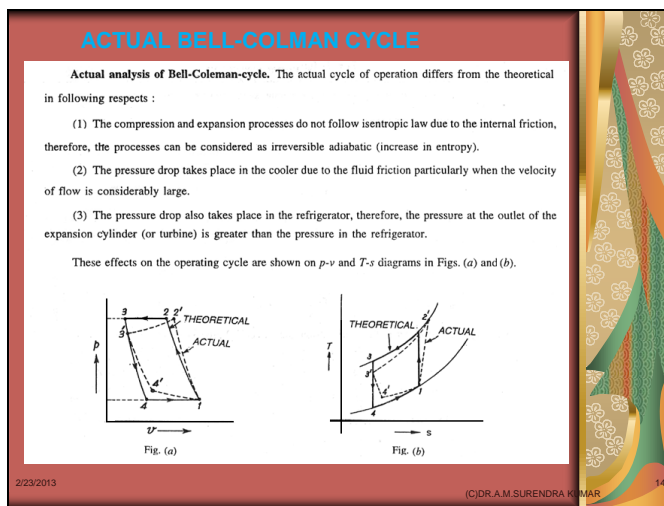
(A) Compressor operation (B) Bell-Coleman cycle (C) Expander operation

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Slide 9



Slide 14



Slide 15

**NECESSITY OF COOLING THE AEROPLANE**

- ❏ Temperature is **low** at higher altitude.
- ❏ But many external & internal heat sources – Add the **heat** in the Cabin
- ❏ For comfort feeling, temperature of air should not exceed above **22°C** – maintain this temp. in the cabin
- ❏ **-5°C** is required for preservation of food & drinks

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Slide 16

### EXTERNAL HEAT SOURCES

- (1) Sun pours heavy quantity of heat in the passenger cabin through the windows and also heats the top surface of the plane which further transfers the heat to the cabin.
- (2) The pressure required in the passenger cabin is atmospheric pressure at sea level so that it is necessary to compress the ambient air from the ambient pressure to the cabin pressure because the pressure of the atmosphere decreases with an increase in altitude. Due to this compression, the temperature of the air entering the cabin will be higher than the ambient temperature.
- (3) When the fast moving plane passes through air, it compresses a layer of air and raises its temperature. Owing to the skin friction, this compression occurs all over the aircraft and high temperature air layer will be always in contact with the external surface of the aircraft, so that, the heat will flow from the surface of the air-plane to the cabin.

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Slide 17

$V_1 = V_a =$  Speed of Aeroplane or speed of air in opposite direction  
 $h_1 =$  Enthalpy of air before compression  
 $h_2 =$  Enthalpy of air after compression  
 $V_2 =$  Velocity of stagnant air.

Applying steady flow energy equation to the flowing air

$$\frac{V_1^2}{2} + h_1 = \frac{V_2^2}{2} + h_2$$


As  $V_2 = 0$

$$\therefore (h_2 - h_1) = C_p (T_2 - T_1) = \frac{V_1^2}{2}$$

$$\therefore \Delta T = (T_2 - T_1) = \text{Rise in temperature} = \frac{V_1^2}{2C_p}$$

The actual rise in temperature will be 80 to 90% of the rise calculated theoretically because of heat transfer from hot compressed air to the colder surrounding.

An aeroplane moving with 1000 km/hour will experience 50°C rise in temperature of the surface.



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Slide 18

### INTERNAL HEAT SOURCES

**Internal Sources.** (1) Human body continuously generates heat at a rate of 400 kJ per hour at rest and this heat is dissipated to the surroundings or to the cabin air. The passengers and pilot give off the heat to the cabin air.

(2) The electrical and electronic equipments used for the control system generate heat and this heat should be taken out for the efficient working of the control system. An aeroplane of 3000 KW capacity carrying 50 – 75 passengers requires control equipments of 10 kW capacity and it requires nearly 3 tons of refrigeration or cooling capacity.

(3) The engine parts of the aeroplane are subjected to a high temperature throughout the flight and heat from these parts comes to the cabin through conduction, convection and radiation.

All the above sources which are adding heat to the cabin will increase the cabin temperature and it becomes uncomfortable for the passengers. In the actual design, proper care is taken to reduce the heat flow from different sources mentioned above. Nowadays, it has become a common trend to equip the aeroplane with the refrigeration plant as the man requires more comfort.

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Slide 19

The advantages of air refrigeration for aeroplane are summarised as follows :

- (1) As the compressed air is already available, there is no necessity to install a separate compression equipment for the system.
- (2) Weight per ton of refrigeration capacity is least.
- (3) Space and volume required per ton to set the plant is less compared with other systems.
- (4) Air being non-flammable, there is no danger of fire.
- (5) Leakage problem does not arise in air-refrigeration system which is the serious problem in vapour compression system.
- (6) Maintenance cost is low.
- (7) Any part of the air refrigeration system can be removed and repaired easily

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Slide 20

### TYPES OF AIR REFRIGERATION SYSTEM

1. Simple cooling cycle system
2. Evaporative cooling system.
3. Boot strap air cycle refrigeration system
4. Regenerative cooling system.
5. Reduced ambient type cooling system

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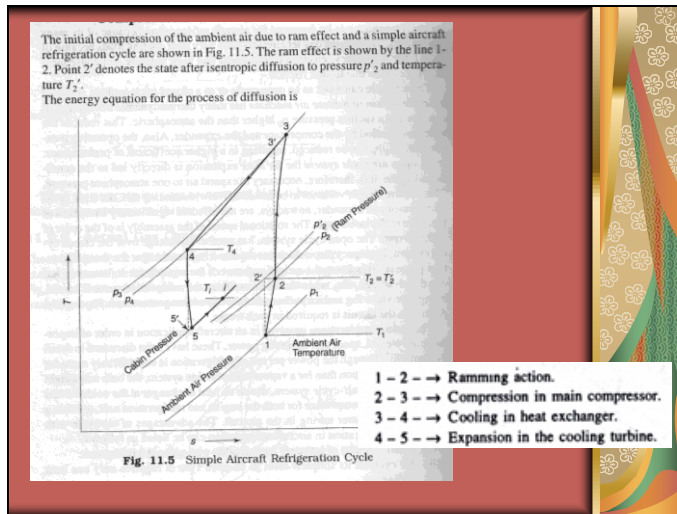
Slide 21

### SIMPLE EVAPORATIVE COOLING TYPE A/C SYSTEM

Fig. (a) Simple cooling cycle.

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Slide 22



Slide 23

Power required for the refrigeration system =  $m_a(\text{kg/s}) \times \Delta h(\text{kJ/kg})$   
 (to run the compressor)

$$= m_a C_p (T_3 - T_2) \text{ kW}$$

C.O.P of the system = refrigerating effect / Work for compressor

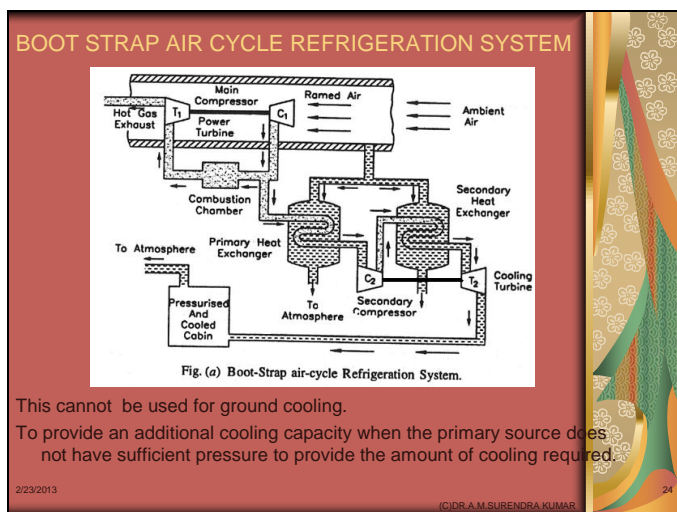
$$= \frac{\text{Refrigerating capacity (in kW)}}{\text{Power (in kW) for Running compressor}}$$

$$= \frac{3.5 \text{ TR}}{\text{Power required for compressor}}$$

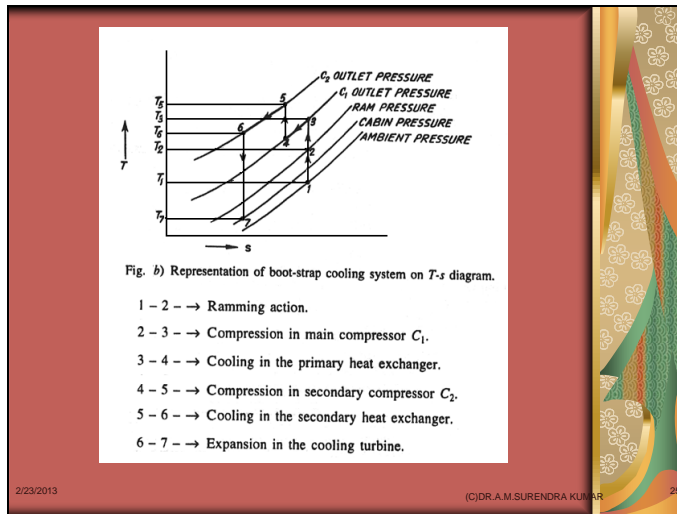
Where TR = tonn of refrigeration = 210 kJ/minute = 3.5 kW

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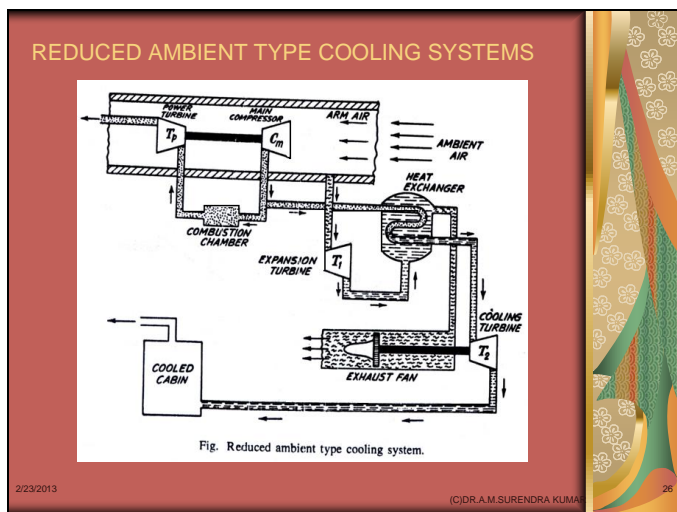
Slide 24



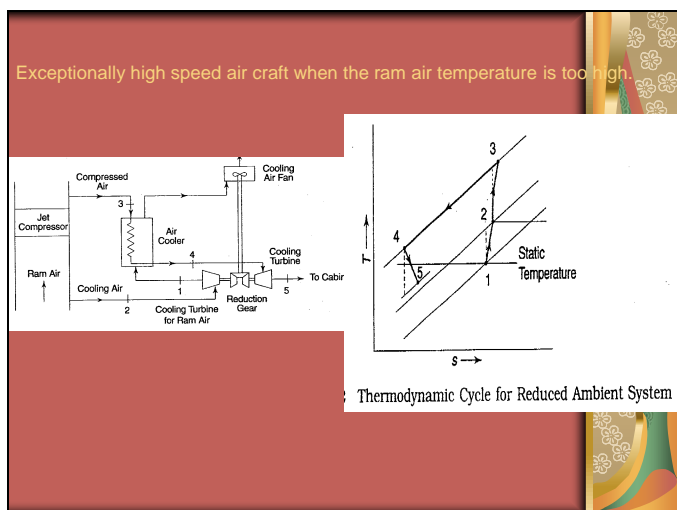
Slide 25



Slide 26



Slide 27





Slide 28

## DRY AIR RATED TEMPERATURE [DART]

A comparison of the different types of air-refrigeration systems is made in terms of *dry air rated temperature* (DART). This is the temperature of the discharge air from the expander if there is no condensed moisture present.

The rating of the aircraft units is given in terms of kg of air per unit time at the design DART. Thus the capacity of the machine giving  $\dot{m}$  kg/s of air at a DART of  $t_0$  to maintain a cabin at temperature  $t_1$  is

$$\dot{Q}_0 = \dot{m} C_p (t_1 - t_0)$$

28

Slide 29

## COMPARISON OF DIFFERENT AIR COOLING SYSTEM USED IN AIR CRAFT

The graph plots Discharge Temperature of Cooling Turbine (°C) on the y-axis (ranging from -60 to 60) against Mech Number on the x-axis (ranging from 0.2 to 2.6). Five curves are shown: 'SIMPLE' (solid line, increasing), 'BOOT-STRAP' (solid line, increasing), 'SIMPLE EVAPORATIVE' (solid line, decreasing), 'REDUCED AMBIENT' (dashed line, decreasing), and 'BOOT-STRAP EVAPORATIVE' (dashed line, decreasing).

Fig. Air cycle cooling system performance curves (approximate).

- (i) The simple system is preferable at low speeds.
- (ii) Above a speed of 1000 kmph, the regenerative system is necessary. The Bootstrap system is also used.
- (iii) Reduced ambient system may be useful for supersonic aircrafts and rockets.

29

Slide 30

## ACTUAL AIR-CONDITIONING SYSTEM WITH CONTROLS

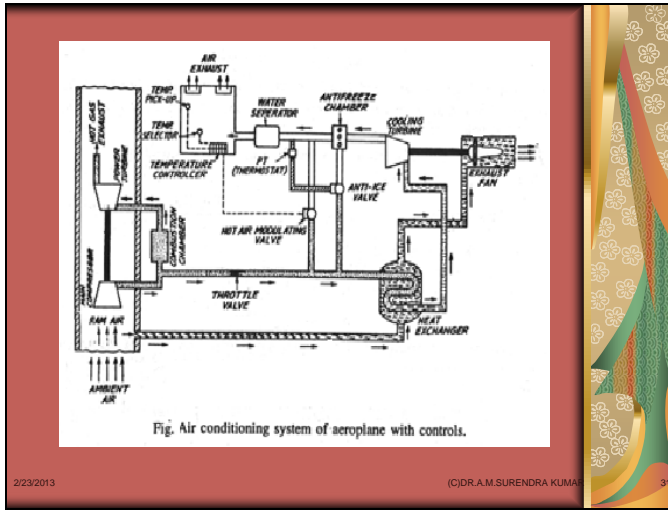
In an automatic control Air-conditioning system, the following three controls are generally incorporated for the efficient working of the A/C system.

1. **Temperature** control in the Cabin
2. **Air flow** (quantity) control in the Cabin.
3. **Water separator** and anti-ice control

30



Slide 31



2/23/2013

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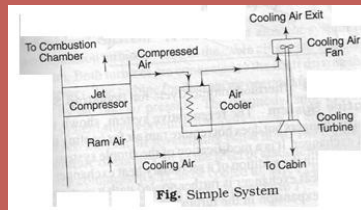
31

Slide 32

**Example 1**

An air-cycle unit operating on the simple system is designed for the following conditions. 7.5 kg/min of air entering the turbine at 4.4 bar, cabin pressure 1 bar, dry air rated discharge temperature of  $-6^{\circ}\text{C}$  and a turbine efficiency of 80 %.

- (a) At what temperature does the air enter the turbine ?
- (b) How much power does the turbine supply to the fan ?



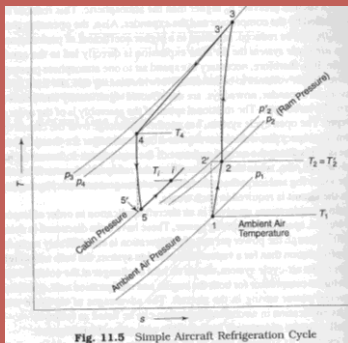
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32

Slide 33

In this system, the compressed air after cooling in air cooler (H.E) is passed through a cooling turbine. The work of this turbine is to drive a fan which draws cooling air through the H.E. This fan is put on the down stream side thus avoiding the additional temperature.



2/23/2013

33

## Slide 34

Given,  $\dot{m} = 7.5 \text{ k/min}$ ,  $P_4 = 4.4 \text{ bar}$ ,  $P_5 = 1 \text{ bar}$ ,  $\eta_{\text{turbine}} = 80\%$

$T_5 = -6^\circ\text{C} = 267 \text{ K}$  Find  $T_4 = ?$  & Turbine power = ?

$$\frac{T_4}{T_5} = \left( \frac{P_4}{P_5} \right)^{\frac{\gamma-1}{\gamma}} = \left( \frac{4.4}{1} \right)^{\frac{1.4-1}{1.4}}$$

$$= 1.52$$

$$T_4 = 1.52 T_5$$

$$\eta_{\text{turbine}} = \frac{\Delta h_{\text{ACTUAL}}}{\Delta h_{\text{ISENTROPIC}}} = \frac{h_4 - h_5}{h_4 - h_{5'}} = \frac{T_4 - T_5}{T_4 - T_5'}$$

$$0.80 = \frac{1.52 T_5' - 267}{1.52 T_5' - T_5'}$$

$$T_5' = 241.84 \text{ K}$$

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## Slide 35

$$T_4 = 1.52 T_5'$$

$$= 1.52 \times 241.84$$

$$= 367 \text{ K}$$

Turbine work =  $m \times c_p \times (T_4 - T_5)$

$$= (7.5/60) \times 1.005 \times (367 - 267)$$

$$= 12.63 \text{ kW}$$

a) The temperature does the air enter the turbine  $T_4 = 367.60 \text{ K} = 94.6^\circ\text{C}$

a) Turbine power supply to the fan = 12.63 kW

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